

Solutions to Exam in FRTN10 Multivariable Control 2018-10-27

1 a. From the block diagram we can easily find that

$$G(s) = \begin{bmatrix} \frac{1}{s+1} & \frac{-2}{s+1} \\ \frac{3}{s+1} & \frac{3(s-1)}{(s+1)(s+3)} \end{bmatrix}$$

b. One can immediately see from the block diagram that the system has two poles, one in -1 and one in -3 .

Alternatively, one can calculate all minors and find the least common denominator. The relevant 1st order subdeterminants are the four non-zero elements

$$\frac{1}{s+1}, \quad \frac{-2}{s+1}, \quad \frac{1}{s+1}, \quad \frac{3(s-1)}{(s+1)(s+3)}$$

and the 2nd order subdeterminant is

$$\frac{9(s+1)}{(s+1)^2(s+3)} = \frac{9}{(s+1)(s+3)}$$

The least common denominator of all subdeterminants is

$$p(s) = (s+1)(s+3).$$

The system therefore has two poles: one in -1 and one in -3 .

The maximal minor is the 2nd order subdeterminant $\frac{9}{(s+1)(s+3)}$; the zero polynomial is just a constant, and therefore the system has no zeros.

c. The transfer function from u_1 to y_2 is $G_{21}(s) = \frac{3}{s+1}$. To find the variance of y_2 , we can either directly calculate

$$V(y_2) = \frac{1}{2\pi} \int_{-\infty}^{\infty} G(i\omega)G(-i\omega)d\omega = \frac{1}{2\pi} \int_{-\infty}^{\infty} \frac{d\omega}{\omega^2 + 1} = \frac{\text{atan}(\infty) - \text{atan}(-\infty)}{2\pi} = \frac{9}{2}$$

or introduce the state-space realization $(A, B, C, D) = (-1, 3, 1, 0)$ and solve the Lyapunov equation for $\Pi = V(y_2)$ as

$$A\Pi + \Pi A^T + BB^T = 0 \Leftrightarrow -2\Pi + 9 = 0 \Leftrightarrow \Pi = \frac{9}{2}$$

2. From the dimensions given (3 poles/states, 1 input, 2 outputs), the A matrix should be 3×3 , the B matrix 3×1 and the C matrix 2×3 . The easiest option is to choose a system in diagonal form with some stable eigenvalues (LHP poles) on the diagonal, for example

$$\dot{x} = \begin{bmatrix} -1 & 0 & 0 \\ 0 & -2 & 0 \\ 0 & 0 & -3 \end{bmatrix} x + \begin{bmatrix} 0 \\ 1 \\ 1 \end{bmatrix} u$$

$$y = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 1 \end{bmatrix} x$$

With a zero in the top B element, we immediately see that the first state is not controllable. Alternatively we can compute the controllability matrix $\mathcal{C} = [B \ AB \ A^2B]$ and note that the first row is all zeros \Rightarrow not full rank.

With y_1 , we directly measure x_1 . With $y_2 = x_2 + x_3$, we can observe both states since they have different eigenvalues. Alternatively we can compute the observability matrix $\mathcal{O} = [C^T \ A^T C^T \ (A^T)^2 C^T]^T$ and see that it has full rank. (Any other C matrix that includes all three states would also work.)

The transfer function has the form $\frac{N_1}{s+1} + \frac{N_2}{s+2} + \frac{N_3}{s+3}$, which is even strictly proper (if $D = 0$). Alternatively, we can directly calculate $G(s)$ and verify that it is proper.

3 a. From the block diagram, we have

$$\begin{aligned} -QPw + R(F(w-v) - GPw) = v & \Rightarrow (I + RF)v = (RF - RGP - QP)w \\ & \Rightarrow v = (I + RF)^{-1}(RF - RGP - QP)w \\ & \Rightarrow H = (I + RF)^{-1}(RF - RGP - QP) \end{aligned}$$

b. To be able to conclude stability of the closed-loop system with the Small Gain Theorem, in addition to having a stable Δ_i , we should have

$$\|\Delta_i\| < \frac{1}{\|H\|_\infty} = 1/4$$

Δ_2 is not stable, so we can not use Small Gain Theorem for it. The other blocks have the maximum gains

$$\|\Delta_1\| = 1/8, \quad \|\Delta_3\| = 1/2, \quad \|\Delta_4\| = 1/5$$

We can hence guarantee stability of the control loop with Δ_1 and Δ_4 .

- 4 a.** Yes. This can be seen in many ways, for instance in that $P/(1 + PC)$ approaches zero for low frequencies, indicating that a constant load disturbance can be rejected. It can also be seen in the difference at low frequencies between $C/(1 + PC)$ and $1/(1 + PC)$.
- b.** Yes. The maximum sensitivity is smaller than 2, indicating a gain margin of at least 2. The maximum complementary sensitivity is even smaller, implying robust stability.
- c.** In the curve for $C/(1 + PC)$ we read out the maximum gain ≈ 9 .
- d.** No. The time delay imposes an upper limit of the achievable bandwidth of about $1.6/L = 16$ rad/s, and the current bandwidth is only somewhat lower than this.
- e.** The curve for $PCF/(1 + PC)$ shows that the step response will be smooth and well-damped, with a bandwidth of about 10 rad/s. The process time delay will also show up in the step response. The static gain is 1.
- 5 a.** Since G is triangular, we immediately know that $RGA = I$. Alternatively, we can calculate the RGA according to

$$G(0) = \begin{bmatrix} 1 & 1 & 0 \\ 0 & 1 & 2 \\ 0 & 0 & -1 \end{bmatrix}, \quad G(0)^{-1} = \begin{bmatrix} 1 & -1 & -2 \\ 0 & 1 & 2 \\ 0 & 0 & -1 \end{bmatrix}$$

$$RGA(G(0)) = G(0) .* G(0)^{-T} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

From this, a suitable pairing is input 1 with output 1, input 2 with output 2 and input 3 with output 3.

This is indeed the only reasonable choice. From the transfer matrix we see that y_3 is only affected by u_3 , so that pairing is given. If u_3 is used for y_3 then u_2 has to be used for y_2 since it's unaffected by u_1 . This leaves u_1 for y_1 . All other decentralized schemes would have an output that couldn't be controlled.

- b. For its input pairing, the last output has the transfer function $P = \frac{1}{s-1}$. We can write out the requirement for first order roll-off as

$$|T(i\omega)| \leq \left| \frac{k}{i\omega} \right| \Leftrightarrow \left| \frac{i\omega}{k} T(i\omega) \right| \leq 1 \Leftrightarrow \left\| \frac{i\omega}{k} T(i\omega) \right\|_{\infty} \leq 1$$

However, we know that the rightmost condition can only be satisfied if $\left| \frac{p}{k} \right| \leq 1$ for all unstable poles p of the process. We have an unstable pole in 1, which gives

$$\frac{1}{k} \leq 1$$

The conclusion is that k cannot be smaller than 1.

- 6 a. From the block diagram (setting $C = 0$) we get

$$P = \begin{bmatrix} 0 & P_0 & 0 & P_0 \\ 1 & P_0 & 0 & P_0 \\ 0 & 0 & 1 & 0 \end{bmatrix}$$

with

$$\begin{aligned} P_{z\omega} &= [0 \quad P_0 \quad 0] & P_{zu} &= [P_0] \\ P_{y\omega} &= \begin{bmatrix} 1 & P_0 & 0 \\ 0 & 0 & 1 \end{bmatrix} & P_{yu} &= \begin{bmatrix} P_0 \\ 0 \end{bmatrix} \end{aligned}$$

- b. We need to add the error to the output of our model, i.e.

$$z = \begin{bmatrix} x \\ x - r \end{bmatrix}$$

resulting in

$$P = \begin{bmatrix} 0 & P_0 & 0 & P_0 \\ 0 & P_0 & -1 & P_0 \\ 1 & P_0 & 0 & P_0 \\ 0 & 0 & 1 & 0 \end{bmatrix}$$

- c.

$$G_{cl} = P_{z\omega} + P_{zu}QP_{y\omega}$$

The Youla parametrization is preferred since the closed-loop transfer function then is linear (actually affine) in the design variable. Without this the resulting optimization problem is much harder to solve.

- 7 a. Noting that A and Q_{12} are zero matrices and Q_2 is the identity results in the following simplification of the Riccati equation:

$$Q_1 - SBB^T S = 0$$

Setting

$$S = \begin{bmatrix} a & c \\ c & b \end{bmatrix}$$

and noting that $BB^T = I$ results in the following system of equations:

$$\begin{aligned} 3 &= a^2 + c^2 \\ 1 &= b^2 + c^2 \\ 0 &= (a+b)c \end{aligned}$$

From the last equation we can conclude that $c = 0$, since otherwise either a or b needs to be negative and a positive definite matrix can't have negative diagonal elements. This results in

$$S = \begin{bmatrix} \sqrt{3} & 0 \\ 0 & 1 \end{bmatrix}$$

and the feedback law is given by

$$u = -(SB)^T x = - \begin{bmatrix} 0 & 1 \\ \sqrt{3} & 0 \end{bmatrix} x$$

- b. The system description with noise is

$$\begin{aligned} \dot{x} &= \begin{bmatrix} 0 & 1 \\ 1 & 0 \end{bmatrix} x + u + w_1 \\ y &= \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} x + w_2 \end{aligned}$$

where w_1 and w_2 are two-dimensional white noise processes with intensities $R_1 = R_2 = I$. Since A and R_{12} are zero and R_1 , R_2 and C are the identity we get the Riccati equation

$$I = PP$$

Setting

$$P = \begin{bmatrix} a & c \\ c & b \end{bmatrix}$$

results in

$$\begin{aligned} 1 &= a^2 + c^2 \\ 1 &= b^2 + c^2 \\ 0 &= (a+b)c \end{aligned}$$

and with similar reasoning as in the previous question we get

$$P = I$$

and the Kalman gain becomes

$$K = PC^T = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

And the Kalman filter is given by

$$\hat{x} = -\hat{x} + \begin{bmatrix} 0 & 1 \\ 1 & 0 \end{bmatrix} u + y$$

- c. The separation principle for LQG control says that the feedback law we calculated in the fully observed case still is optimal in the uncertain case.
- d. A reasonable suggestion for the source of the high-frequency components would be high-frequency measurement noise. In order for the Kalman filter to take greater consideration of this noise we need to model it. Therefore, scaling R_2 to make it larger (or scaling R_1 to make it smaller) would be a sensible first step in the tuning process.