



FRTN10 Multivariable Control, Lecture 12

Automatic Control LTH, 2017

Course Outline

- L1-L5 Specifications, models and loop-shaping by hand
- L6-L8 Limitations on achievable performance
- L9-L11 Controller optimization: Analytic approach
- L12-L14 Controller optimization: Numerical approach
 - T2 **Youla parameterization, internal model control**
 - T3 Synthesis by convex optimization
 - T4 Controller simplification

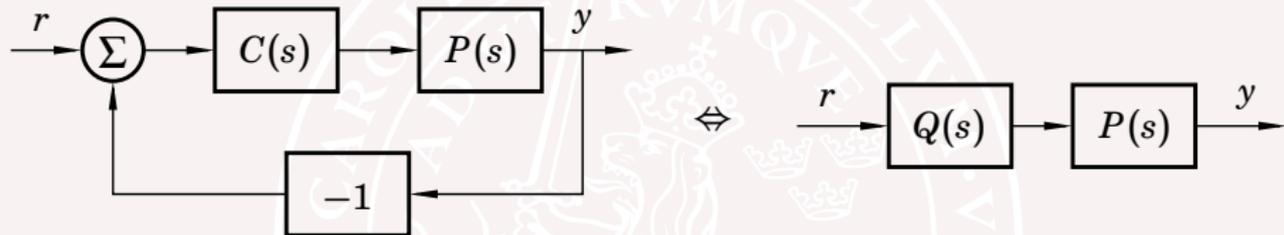
Lecture 12 – Outline

- 1 The Quola parameterization
- 2 Internal model control (IMC)

[Glad&Ljung Section 8.4]

Basic idea of Youla and IMC

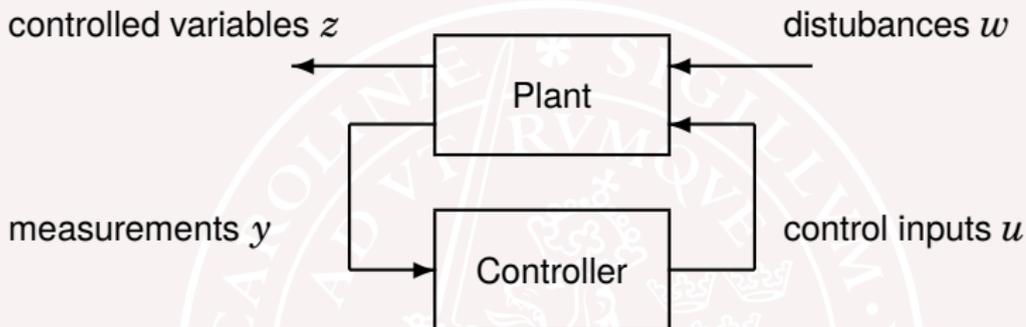
Assume stable SISO plant P . Model for design:



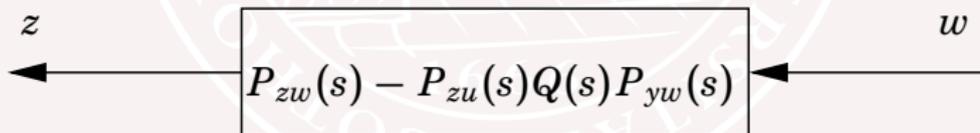
$$\frac{PC}{1 + PC} = PQ$$
$$Q = \frac{C}{1 + PC}$$

Design Q to get desired closed-loop properties. Then $C = \frac{Q}{1 - QP}$

General idea for Lectures 12–14



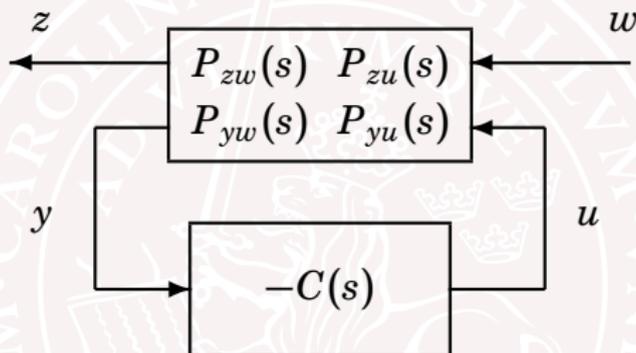
The choice of controller corresponds to designing a transfer matrix $Q(s)$, to get desirable properties of the following map from w to z :



Once $Q(s)$ has been designed, the corresponding controller can be found.

The Youla (Q) parameterization

General closed-loop control system:

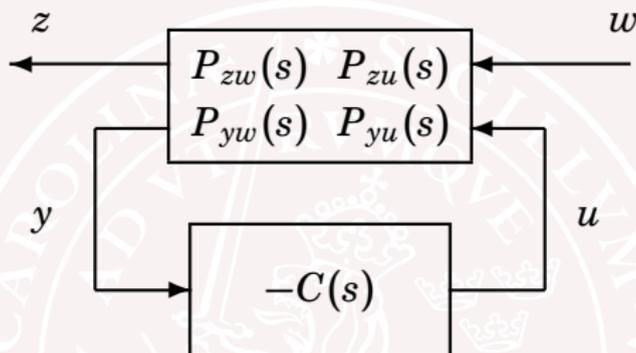


$$Z(s) = P_{zw}(s)W(s) + P_{zu}(s)U(s)$$

$$Y(s) = P_{yw}(s)W(s) + P_{yu}(s)U(s)$$

$$U(s) = -C(s)Y(s)$$

The Youla (Q) parameterization



Closed-loop transfer function from w to z :

$$G_{zw}(s) = P_{zw}(s) - P_{zu}(s) \underbrace{C(s) [I + P_{yu}(s)C(s)]^{-1}}_{=Q(s)} P_{yw}(s)$$

Given $Q(s)$, the controller is $C(s) = [I - Q(s)P_{yu}(s)]^{-1}Q(s)$

All stabilizing controllers

Suppose the plant $P = \begin{bmatrix} P_{zw} & P_{zu} \\ P_{yw} & P_{yu} \end{bmatrix}$ is stable. Then

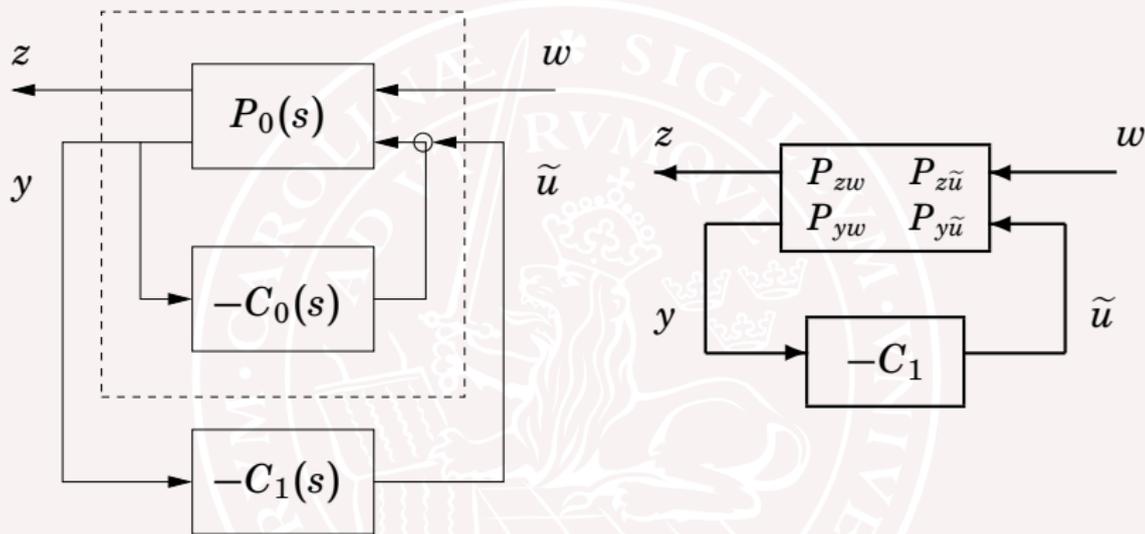
- Stability of Q implies stability of $P_{zw} - P_{zu}QP_{yw}$
- If $Q = C[I + P_{yu}C]^{-1}$ is unstable, then the closed loop is unstable.

Hence, if P is stable then **all stabilizing controllers** are given by

$$C(s) = [I - Q(s)P_{yu}(s)]^{-1}Q(s)$$

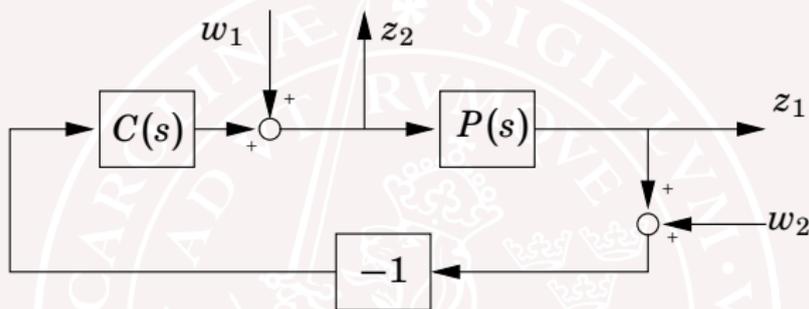
where $Q(s)$ is an arbitrary stable transfer function.

Dealing with unstable plants



If $P_0(s)$ is unstable, let $C_0(s)$ be some stabilizing controller. Then the previous argument can be applied with P_{zw} , $P_{z\tilde{u}}$, P_{yw} , and $P_{y\tilde{u}}$ representing the stabilized system.

Example – DC-motor



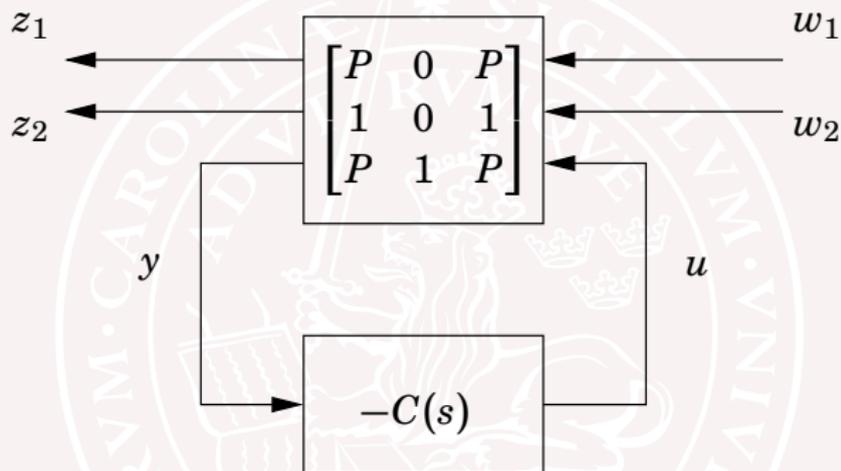
Assume we want to optimize the closed-loop transfer matrix from $(w_1, w_2)^T$ to $(z_1, z_2)^T$,

$$G_{zw}(s) = \begin{bmatrix} \frac{P}{1+PC} & \frac{-PC}{1+PC} \\ \frac{1}{1+PC} & \frac{-C}{1+PC} \end{bmatrix}$$

when $P(s) = \frac{20}{s(s+1)}$. How to parameterize all stabilizing controllers $C(s)$?

Stabilizing controller for DC-motor

Generalized plant model:

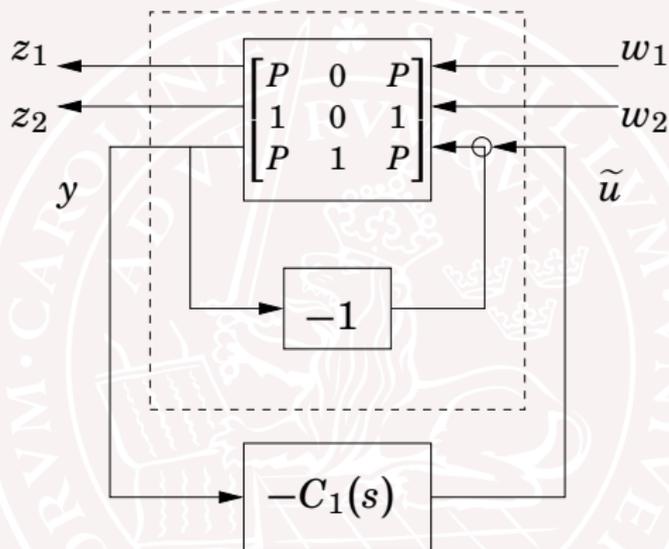


$P(s) = \frac{20}{s(s+1)}$ is not stable, so introduce

$$C(s) = C_0(s) + C_1(s)$$

where $C_0(s) = 1$ stabilizes the plant; $P_c(s) = \frac{P(s)}{1+P(s)} = \frac{20}{s^2+s+20}$

Redrawn diagram for DC-motor example

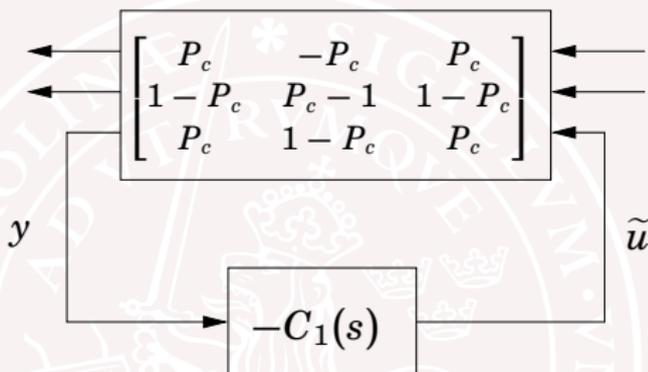


$$z_1 = Pw_1 + P(\tilde{u} - y)$$

$$z_2 = w_1 + \tilde{u} - y$$

$$y = Pw_1 + w_2 + P(\tilde{u} - y) \Rightarrow y = \frac{P}{1+P}w_1 + \frac{1}{1+P}w_2 + \frac{P}{1+P}\tilde{u}$$

Redrawn diagram for DC-motor example



$$P_{y\tilde{u}} = P_c$$

$$G_{zw} = \underbrace{\begin{bmatrix} P_c & -P_c \\ 1 - P_c & P_c - 1 \end{bmatrix}}_{P_{zw}} - \underbrace{\begin{bmatrix} P_c \\ 1 - P_c \end{bmatrix}}_{P_{z\tilde{u}}} Q \underbrace{\begin{bmatrix} P_c & 1 - P_c \end{bmatrix}}_{P_{yw}}$$

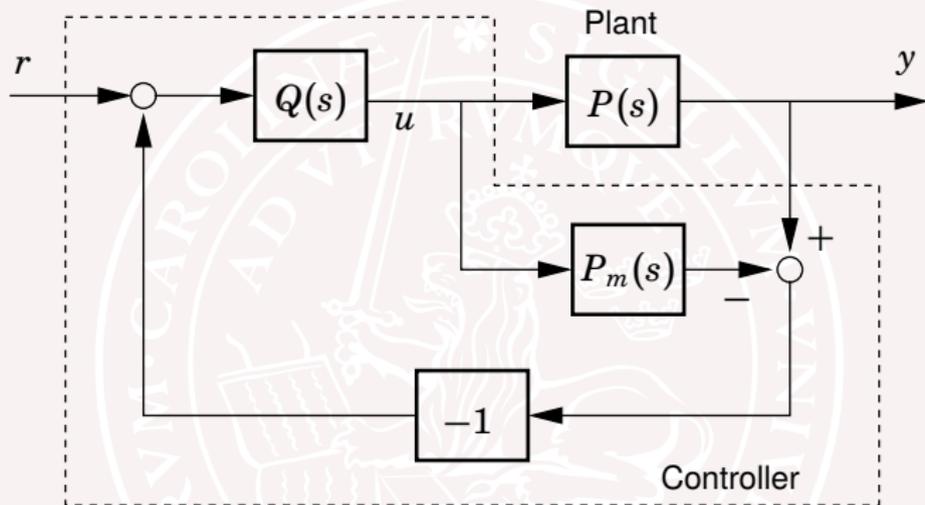
Apply optimization (Lec. 13) to find $Q(s)$. Then $C(s) = 1 + \frac{Q(s)}{1 - Q(s)P_{y\tilde{u}}(s)}$

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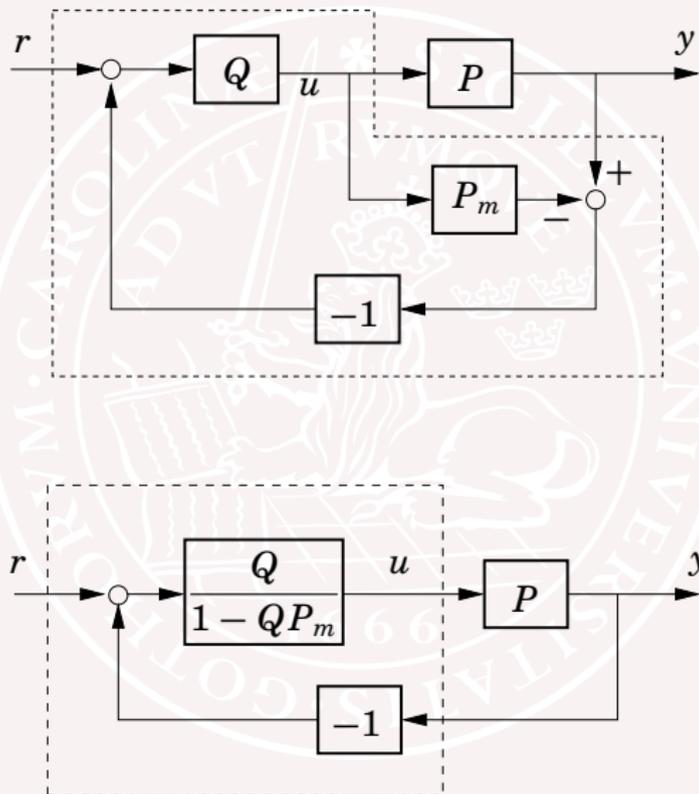
Internal model control (IMC)



Feedback is used only if the real plant $P(s)$ deviates from the model $P_m(s)$. $Q(s)$, $P(s)$, $P_m(s)$ must be stable.

If $P_m(s) = P(s)$, the transfer function from r to y is $P(s)Q(s)$.

Two equivalent diagrams



IMC design rules

When $P = P_m$, the transfer function from r to y is $P(s)Q(s)$.

For perfect reference following, one would like to have $Q(s) = P^{-1}(s)$, but that is not possible (why?)

Design rules:

- If $P(s)$ is strictly proper, the inverse would have more zeros than poles. Instead, one can choose

$$Q(s) = \frac{1}{(\lambda s + 1)^n} P^{-1}(s)$$

where n is large enough to make Q proper. The parameter λ determines the speed of the closed-loop system.

(cf. feedforward design in Lecture 4)

IMC design rules

- If $P(s)$ has unstable zeros, the inverse would be unstable. Options:
 - Remove every unstable factor $(-\beta s + 1)$ from the plant numerator before inverting.
 - Replace every unstable factor $(-\beta s + 1)$ with $(\beta s + 1)$. With this option, only the phase is modified, not the amplitude function.
- If $P(s)$ includes a time delay, its inverse would have to predict the future. Instead, the time delay is removed before inverting.

IMC design example 1 — first-order plant

$$P(s) = \frac{1}{\tau s + 1}$$

$$Q(s) = \frac{1}{\lambda s + 1} P(s)^{-1} = \frac{\tau s + 1}{\lambda s + 1}$$

$$C(s) = \frac{Q(s)}{1 - Q(s)P(s)} = \frac{\frac{\tau s + 1}{\lambda s + 1}}{1 - \frac{1}{\lambda s + 1}} = \underbrace{\frac{\tau}{\lambda} \left(1 + \frac{1}{s\tau} \right)}_{\text{PI controller}}$$

Note that $T_i = \tau$

This way of tuning a PI controller is known as *lambda tuning*

IMC design example 2 — non-minimum phase plant

$$P(s) = \frac{-\beta s + 1}{\tau s + 1}$$

$$Q(s) = \frac{(-\beta s + 1)}{(\beta s + 1)} P(s)^{-1} = \frac{\tau s + 1}{\beta s + 1}$$

$$C(s) = \frac{Q(s)}{1 - Q(s)P(s)} = \frac{\frac{\tau s + 1}{\beta s + 1}}{1 - \frac{(-\beta s + 1)}{(\beta s + 1)}} = \underbrace{\frac{\tau}{2\beta} \left(1 + \frac{1}{s\tau} \right)}_{\text{PI controller}}$$

Note that, again, $T_i = \tau$

The gain is adjusted in accordance with the fundamental limitation imposed by the RHP zero in $1/\beta$.

IMC design for dead-time processes

Consider the plant model

$$P_m = P_{0m} e^{-s\tau}$$

where the deadtime τ is assumed known and constant.

Let $C_0 = Q/(1 - QP_{0m})$ be a controller designed for the delay-free plant model P_{0m} . Then

$$Q = \frac{C_0}{1 + C_0 P_{0m}}$$

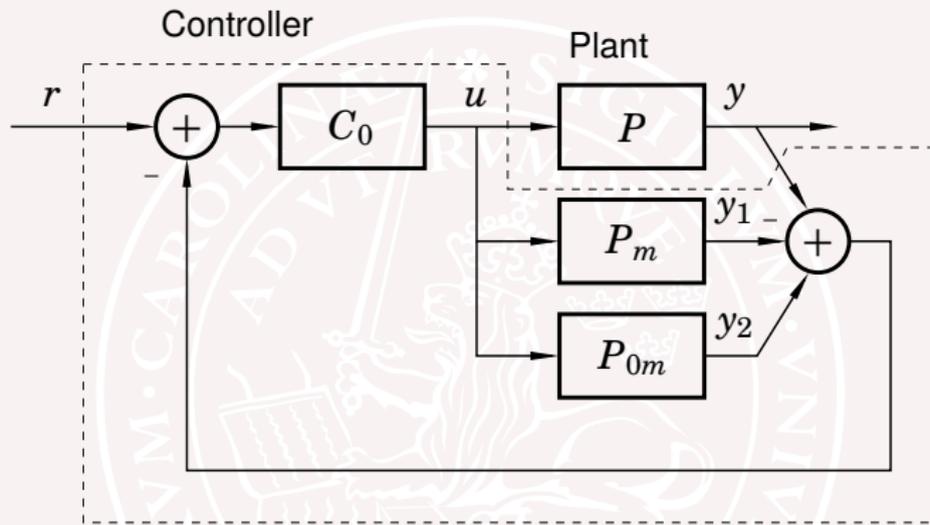
The rule of thumb tell us to use the same Q also for systems with delays.

This gives

$$C = \frac{Q}{1 - QP_{0m}e^{-s\tau}} = \frac{C_0}{1 + (1 - e^{-s\tau})C_0P_{0m}}$$

This modification of C_0 to account for a time delay is known as a Smith predictor.

Smith predictor



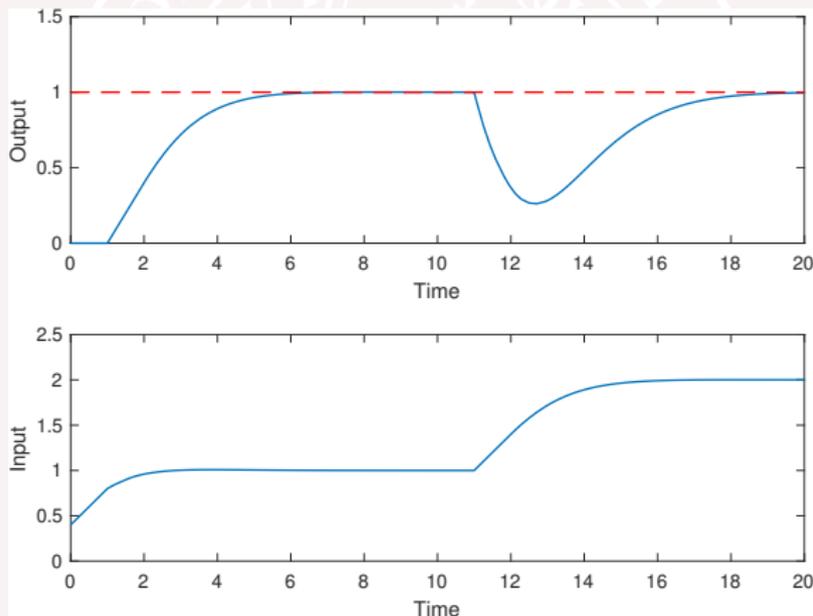
Ideally y and y_1 cancel each other and only feedback from y_2 “without delay” is used. If $P = P_m$ then

$$Y(s) = \frac{C_0(s)P_{0m}(s)}{1 + C_0(s)P_{0m}(s)} e^{-s\tau} R(s)$$

Example

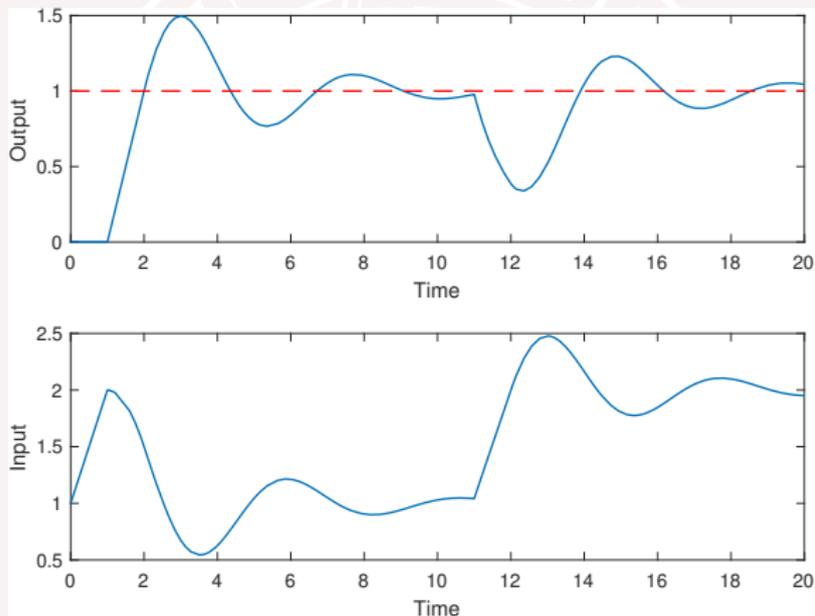
Plant: $P(s) = \frac{1}{s+1}e^{-s}$, nominal controller: $C_0(s) = K \left(1 + \frac{1}{s}\right)$

Simulation with $K = 0.4$, no Smith predictor:



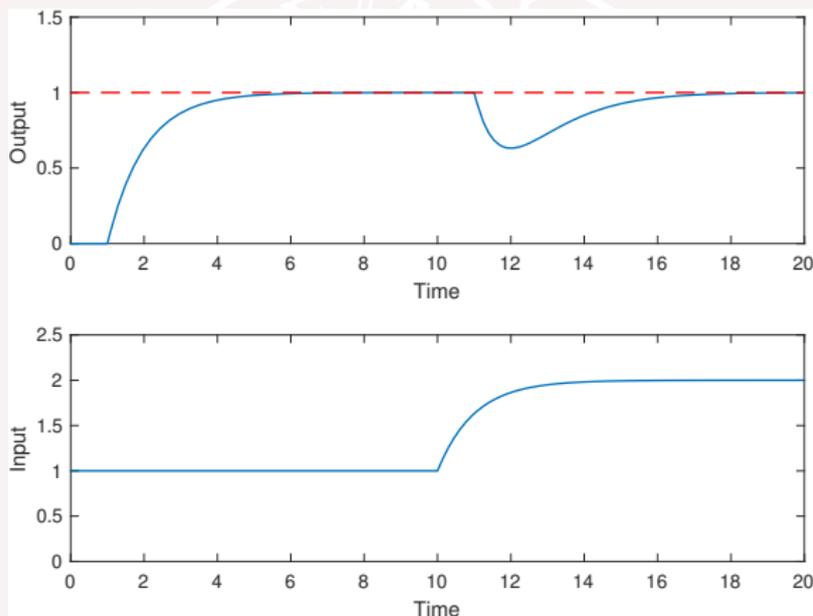
Example

Simulation with $K = 1$, no Smith predictor:



Example

Simulation with $K = 1$ with Smith predictor:



(But do not forget the fundamental limitation imposed by the time delay!)

Lecture 12 – summary

- Idea: Parameterize the closed loop as

$$G_{yr} = PQ \quad \text{SISO case, for IMC design}$$

or

$$G_{zw} = P_{zw} - P_{zu}QP_{yw} \quad \text{General MIMO case, suitable for optimization}$$

for some stable Q .

- After designing Q , the controller is given by

$$C = \frac{Q}{1 - QP} \quad \text{SISO case}$$

or

$$C = [I - QP_{yu}]^{-1}Q \quad \text{General MIMO case}$$